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#### **Nonlinear Model Predictive Control for Preview-Based Traction Control**

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#### Introduction



Vehicle connectivity – V2X



#### Introduction





Our study presents a nonlinear model predictive control (NMPC) formulation for preview-based traction control, which uses the information on the expected tire-road friction coefficient ahead to enhance the wheel slip control performance.



#### Introduction



- Proof-of-concept experiments on an electric vehicle prototype highlight the real-time capability of the controller, and the wheel slip control performance improvement brought by the tire-road friction coefficient preview.
- An experimentally validated simulation model is used in sensitivity analyses, to evaluate the
  performance benefit of the preview-based controller for different powertrain characteristics, such
  as time constants and/or pure time delays.











#### **Control architecture**



Two traction controllers were developed:

- a pre-emptive NMPC with preview of the tire-road friction ahead in the context of V2X;
- a benchmarking non-pre-emptive NMPC, which is only aware of the instantaneous tire-road friction condition.





#### **NMPC formulation**



Continuous time  
formulation  
$$\dot{x}(t) = f(x(t), u(t))$$

States  

$$\mathbf{x} = [T_{m,F}, \omega_{FL}, \omega_{FR}, s_{FL}, s_{FR}, e_{int,FL}, e_{int,FR}]$$

Control action  $\boldsymbol{u} = \left[ T_{m,F,mod}, \varepsilon_{\sigma_{x,FL}}, \varepsilon_{\sigma_{x,FR}} \right]$ 

$$\dot{T}_{m,F} = \frac{T_{m,F,mod} - T_{m,F}}{\tau}$$

$$T_{wh,F,i} - F_{r,F,i}R_{F}$$

$$\dot{\omega}_{Fj} = \frac{T_{Wh,Fj} - T_{X,Fj}R_F}{J_{\omega,F}}$$
$$\dot{s}_{Fj} = \dot{\omega}_{Fj}R_F - \dot{V} = \left[-\frac{R_F^2}{J_{\omega,F}} - \frac{2}{m}\right]F_{X,Fj} + \frac{T_{Wh,Fj}R_F}{J_{\omega,F}}$$
$$\dot{e}_{int,Fj} = e_{Fj} = \sigma_{X,ref,Fj} - \sigma_{X,Fj}$$

$$s_{Fj} = \omega_{Fj}R_F - V$$
  $\sigma_{x,Fj} = \frac{s_{Fj}}{\omega_{Fj}R_F}$ 

$$\dot{V} = \frac{F_{x,FL} + F_{x,FR}}{m} \approx \frac{2F_{x,Fj}}{m}$$

Pacejka magic formula  

$$F_{x,Fj} = \mu_{x,Fj}F_{z,Fj}$$
  
 $\mu_{x,Fj} = D_{Fj}\sin(C_{0,F}\tan^{-1}(B_{Fj}\sigma_{x,Fj}))$   
 $B_{Fj} = B_{0,F}/\mu_{Fj,fut}$   
 $D_{Fj} = D_{0,F}\mu_{Fj,fut}$ 



Reference slip ratio along the prediction horizon  $\sigma_{x,ref,Fj} = f_{\sigma_{x,ref,F}} (\mu_{Fj,fut}, F_{z,Fj})$ 

$$\mu_{Fj,fut}(k) = f_{\mu_{Fj,fut}} [S_{fut}(k) + \Delta x_{delay}(k)\mathbf{1}]$$
$$S_{fut}(k) = S(k)\mathbf{1} + V(k) [t_{fut} - t_{fut,0}\mathbf{1}] \qquad \Delta x_{delay}(k) = V(k)\Delta t_{delay}(k)$$

For the non-pre-emptive NMPC implementation,  $\mu_{Fj,fut}$  is a vector of identical components, equal to the current tireroad friction condition, and  $\Delta t_{delay} = \Delta x_{delay} = 0$ .

Constraints  

$$0 \leq T_{m,F,mod,n} \leq T_{m,F,driver}$$

$$e_{Fj,n} + \varepsilon_{\sigma_{x,Fj},n} \geq 0$$

$$\varepsilon_{\sigma_{x,Fj},n} \geq 0$$

Controller parameter	Experiments	Simulations
Prediction horizon $H_P$ (ms)	250	250
Time step $T_s$ (ms)	25	5
No. of steps N	10	50
Integration time step (ms)	1	1

#### **Nonlinear optimal control problem**



• A generic nonlinear optimal control problem can be defined as the minimization of the following cost function in discrete time form:

$$\min_{\boldsymbol{u}} J(\boldsymbol{x}(0), \boldsymbol{u}(\cdot)) \coloneqq \sum_{n=0}^{N-1} l(\boldsymbol{x}_n, \boldsymbol{u}_n)$$

Where the stage cost function associated with each time step is:

$$l(\mathbf{x}_{n}, \mathbf{u}_{n}) = W_{u,\varepsilon_{\sigma_{x,FL}}}\varepsilon_{\sigma_{x,FL},n}^{2} + W_{u,\varepsilon_{\sigma_{x,FR}}}\varepsilon_{\sigma_{x,FR},n}^{2} + W_{u,T_{m,F}}[T_{m,F,driver} - T_{m,F,mod,n}]^{2} + W_{u,e_{int,FL}}e_{int,FL,n}^{2} + W_{u,e_{int,FR}}e_{int,FR,n}^{2}$$



#### **Vehicle experiments**









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#### **Simulation analysis**



• Simulation-based sensitivity analyses were conducted to evaluate the performance of the preview-based NMPC for different characteristics of the electric powertrain, in terms of: i) motor time constant  $\tau$ ; and ii) pure time delay  $\Delta t_{delay}$  from the torque request to the corresponding torque response.





#### **Simulation analysis results**



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• Effect of the powertrain time constant ( $\Delta t_{delay} = 0$ )



• Effect of the pure time delay ( $\tau = 140 \text{ ms}$ )



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#### **Simulation analysis**

SURREY OF

Overall trends







### The main conclusions are:

- The NMPC can run in real-time on the EV prototype
- The traction controller can pre-emptively reduce wheel torque
- The preview NMPC can compensate for a wide range of dynamic characteristics of the electric powertrains

# THANK YOU





